



بارم هر سوال ۲/۸۰ می باشد.

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الف- به متن درس در فصل ۵ توجه شود.

-ب

$$Q = f\left(R, \mu, \frac{dp}{dx}\right) \text{ four variables } (n = 4)$$

Make a list of the dimensions of these variables from Table 5.1 using the $\{MLT\}$ system:

Q	R	μ	dp/dx
$\{L^3T^{-1}\}$	$\{L\}$	$\{ML^{-1}T^{-1}\}$	$\{ML^{-2}T^{-2}\}$

There are three primary dimensions (M, L, T), hence $j \leq 3$. By trial and error we determine that R, μ , and dp/dx cannot be combined into a pi group. Then $j = 3$, and $n - j = 4 - 3 = 1$. There is only *one* pi group, which we find by combining Q in a power product with the other three:

$$\begin{aligned} \Pi_1 &= R^a \mu^b \left(\frac{dp}{dx}\right)^c Q^1 = (L)^a (ML^{-1}T^{-1})^b (ML^{-2}T^{-2})^c (L^3T^{-1}) \\ &= M^0 L^0 T^0 \end{aligned}$$

Equate exponents:

$$\text{Mass:} \quad b + c = 0$$

$$\text{Length:} \quad a - b - 2c + 3 = 0$$

$$\text{Time:} \quad -b - 2c - 1 = 0$$

Solving simultaneously, we obtain $a = -4$, $b = 1$, and $c = -1$. Then

$$\Pi_1 = R^{-4} \mu^1 \left(\frac{dp}{dx}\right)^{-1} Q$$

or

$$\Pi_1 = \frac{Q\mu}{R^4(dp/dx)} = \text{const}$$



$$\text{HGL}_1 = z_1 + \frac{p_1}{\rho g} = 0 + \frac{350,000}{900(9.807)} = 39.65 \text{ m}$$

$$\text{HGL}_2 = z_2 + \frac{p_2}{\rho g} = 6.43 + \frac{250,000}{900(9.807)} = 34.75 \text{ m}$$

The HGL is lower at section 2; hence the flow is up from 1 to 2 as assumed. *Ans. (a)*

The head loss is the change in HGL:

$$h_f = \text{HGL}_1 - \text{HGL}_2 = 39.65 \text{ m} - 34.75 \text{ m} = 4.9 \text{ m} \quad \text{Ans. (b)}$$

Half the length of the pipe is quite a large head loss.

We can compute Q from the various laminar flow formulas, notably Eq. (6.12):

$$Q = \frac{\pi \rho g d^4 h_f}{128 \mu L} = \frac{\pi (900)(9.807)(0.06)^4 (4.9)}{128(0.18)(10)} = 0.0076 \text{ m}^3/\text{s} \quad \text{Ans. (c)}$$

Divide Q by the pipe area to get the average velocity:

$$V = \frac{Q}{\pi R^2} = \frac{0.0076}{\pi (0.03)^2} = 2.7 \text{ m/s} \quad \text{Ans. (d)}$$

With V known, the Reynolds number is

$$\text{Re}_d = \frac{Vd}{\nu} = \frac{2.7(0.06)}{0.0002} = 810 \quad \text{Ans. (e)}$$

This is well below the transition value $\text{Re}_d = 2300$, so we are fairly certain the flow is laminar.

Notice that by sticking entirely to consistent SI units (meters, seconds, kilograms, newtons) for all variables we avoid the need for any conversion factors in the calculations.



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الف-

$$u(x, y) \approx U \left(\frac{2y}{\delta} - \frac{y^2}{\delta^2} \right) \quad 0 \leq y \leq \delta(x) \quad (7.6)$$

which makes it possible to estimate both momentum thickness and wall shear:

$$\theta = \int_0^\delta \left(\frac{2y}{\delta} - \frac{y^2}{\delta^2} \right) \left(1 - \frac{2y}{\delta} + \frac{y^2}{\delta^2} \right) dy \approx \frac{2}{15} \delta$$

$$\tau_w = \mu \left. \frac{\partial u}{\partial y} \right|_{y=0} \approx \frac{2\mu U}{\delta} \quad (7.7)$$

By substituting (7.7) into (7.5) and rearranging we obtain

$$\delta d\delta \approx 15 \frac{\nu}{U} dx \quad (7.8)$$

where $\nu = \mu/\rho$. We can integrate from 0 to x , assuming that $\delta = 0$ at $x = 0$, the leading edge:

$$\frac{1}{2} \delta^2 = \frac{15\nu x}{U}$$

or

$$\frac{\delta}{x} \approx 5.5 \left(\frac{\nu}{Ux} \right)^{1/2} = \frac{5.5}{\text{Re}_x^{1/2}} \quad (7.9)$$

-ب-

$$B_{net} = (\rho_{air} - \rho_{He}) g \frac{\pi}{6} D^3 = (1.2 - 0.197)(9.81) \frac{\pi}{6} (0.5)^3 \approx 0.644 \text{ N}$$

The net upward force is thus $F_z = (B_{net} - W) = 0.644 - 0.2 = 0.444 \text{ N}$. The balloon drag does depend upon velocity. At 5 m/s, we expect laminar flow:

$$(a) U = 5 \frac{m}{s}: \text{Re}_D = \frac{1.2(5)(0.5)}{1.8E-5} = 167000; \text{ Table 7.3: } C_D \approx 0.47$$

$$\text{Drag} = C_D \frac{\rho}{2} U^2 \frac{\pi}{4} D^2 = 0.47 \left(\frac{1.2}{2} \right) (5)^2 \frac{\pi}{4} (0.5)^2 \approx 1.384 \text{ N}$$

$$\text{Then } \theta_a = \tan^{-1} \left(\frac{\text{Drag}}{F_z} \right) = \tan^{-1} \left(\frac{1.384}{0.444} \right) = 72^\circ \quad \text{Ans. (a)}$$



کد سری سؤال: یک (۱)

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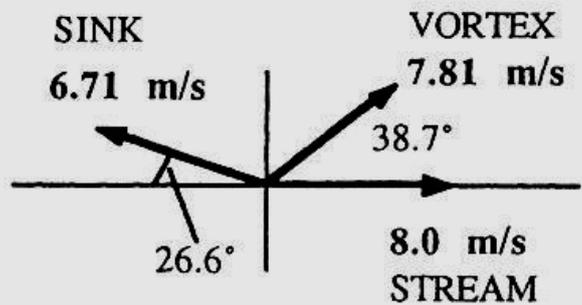
۴- الف- گردش برابر صفر است و جریان غیر چرخشی است.

ب-

Solution: The velocities caused by each term—stream, vortex, and sink—are shown at right. They have to be added together vectorially to give the final result:

$$V = 11.3 \frac{\text{m}}{\text{s}} \text{ at } \theta = 44.2^\circ \angle \text{ Ans.}$$

Fig. P8.23



۵- به توضیحات فصل ۹ کتاب توجه شود.